

Lessons Learned in Achieving Data Center Efficiency Goals – 3 Case Studies

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Developing Innovative, Replicable Solutions with Market Leaders

Better Buildings Challenge

- Better Buildings Alliance
- Better Buildings, Better Plants

Better Buildings Accelerators

- Better Buildings Residential
- Strategic Energy Management

State, Local, and Federal Governments Leading by Example

- Better Communities Alliance
- Performance
 Contracting



Making Energy Efficiency Investment Easier

- Better Buildings Solution Center
- ► Financing Navigator
- Improved Data Consistency and Access
- Tools to Assess the Efficiency of Buildings/Homes
- Tools for Energy Management



U.S. DEPARTMENT OF

Better Buildings and Data Centers

Challenge partners set 10-year, 20% energy-savings targets across portfolio of buildings and share results

Better Buildings Challenge CenturyLink Digital Realty Trust

eBay Inc.

Facebook

Intel Corporation

Intuit

IO Data Centers

Iron Mountain Data Centers

Michigan State University

Sabey Data Center Partners

Staples

Accelerator partners set 5-year, 25% energysavings targets across one or more data centers and provide regular updates on progress

| Better Buildings Accelerator | | |
|------------------------------|-----------------------|--|
| ANL | Defense Health Agency | |
| EMSL | Georgia Tech | |
| Indiana University | LBNL | |
| LLNL | LANL | |
| NERSC | NREL | |
| ORNL | Stanford | |
| State of Michigan | Home Depot | |
| DOD-DISA | DOJ-DEA | |
| VA | EPA | |
| NASA | SSA | |
| University of Colorado | University of Iowa | |
| Virtustream | Waste Management | |

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Data Center Path to Efficiency and Automation

October 2017

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Data Center Path to Efficiency and Automation

- 1. PayPal's data center landscape
- 2. The efficiency challenge
- 3. Our journey so far
- 4. Phase 1 Improving space and power efficiency
- 5. Phase 2 Improving process and tool efficiency
- 6. Phase 3 Planned future improvements

PayPal's data center landscape

PayPal has grown rapidly over the last decade and our data center landscape has become increasingly complex. A key challenge was to maximize efficiency to ensure available capacity.



The efficiency challenge

We captured an initial baseline to determine our energy usage and identify several sources of inefficiencies

- Lack of transparency on Always-on cooling equipment ownership **General Space Energy Use** 2% Lack of proper No mechanism for clear containment Data Center utilization showback **Cooling Energy** No temperature control 24% Require a large buffer in **Protected IT Load** zones capacity to ensure we meet 58% **Electrical Loss Energy** business demand Inefficient CRAH motors • 16% Over-allocation for failure scenarios Significant amount of unused
 - Older transformer-based UPSs

hardware (i.e. zombie assets). Estimated utilization is ~40-50%

Our ultimate goal is to reach 70% true utilization for space and power in the data center and this requires tackling efficiency on multiple fronts

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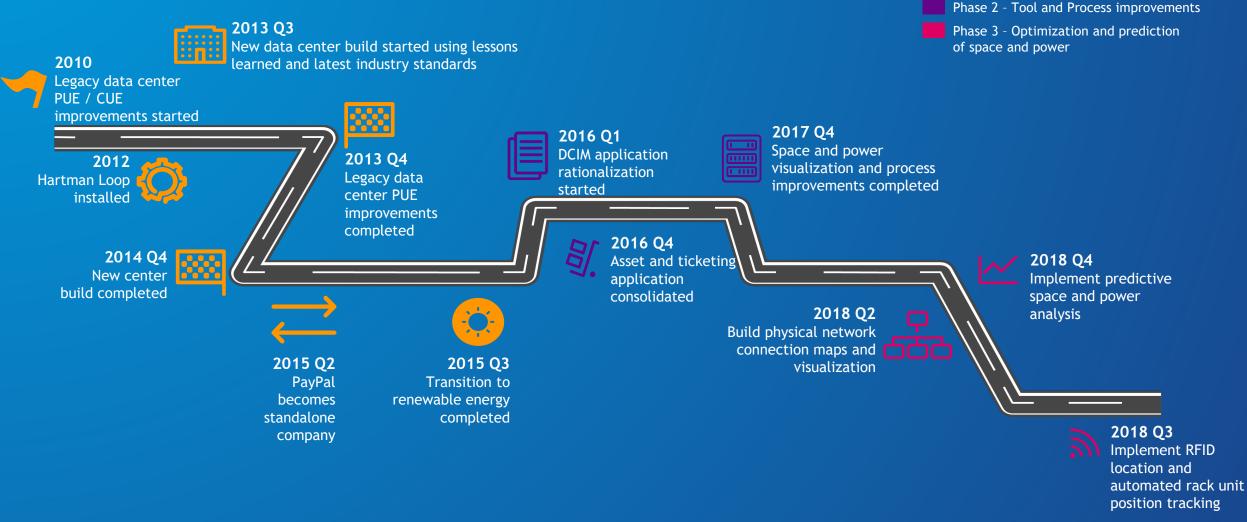
Video

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Our journey so far

We tackled efficiency improvements in multiple phases moving from physical space and power efficiency to people, process and tools.

Phase 1 - PUE and CUE improvements



PayPal

Phase 1 - Improving space and power efficiency We began improving efficiency by tackling quick wins in our legacy Phoenix data center

Current Situation Reducing Data Center cooling energy

- The data center was operated as a standard tier IV facility
- The electrical losses and IT load required long lead times to address

Solution Three year energy management plan

Variable Speed Drives and new high efficiency motors on the CRAH systems

Cold Aisle

Containment

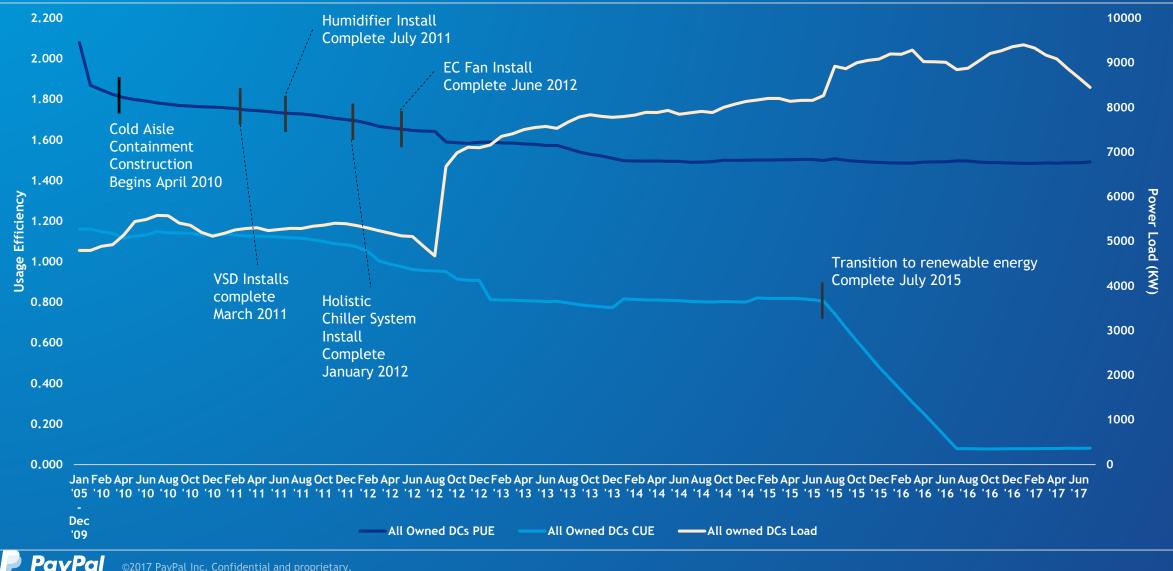
Harmonic Mitigating Transformers in the CRAH electrical distribution system

Hartman loop

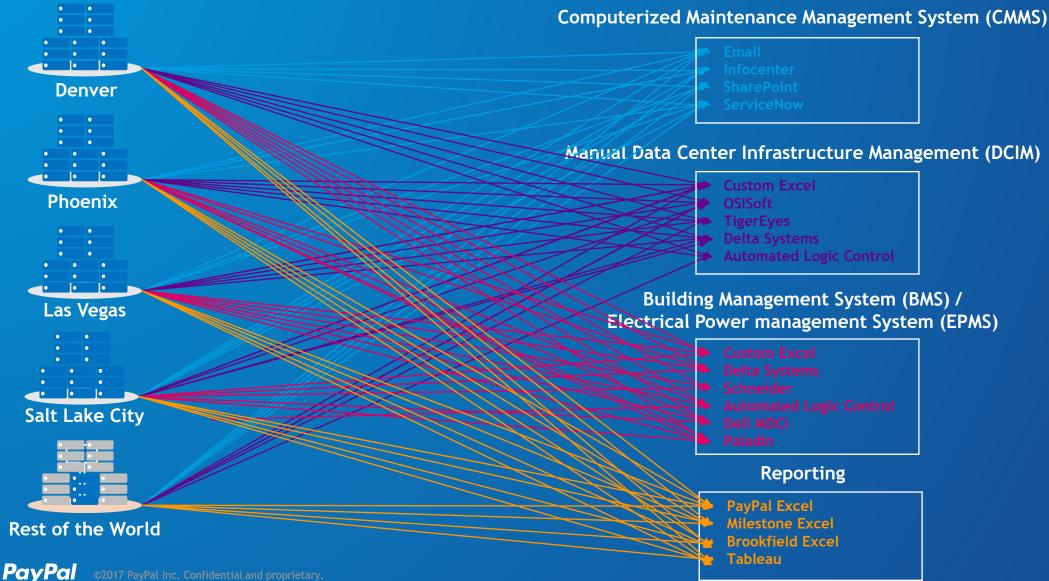
chiller plant optimization

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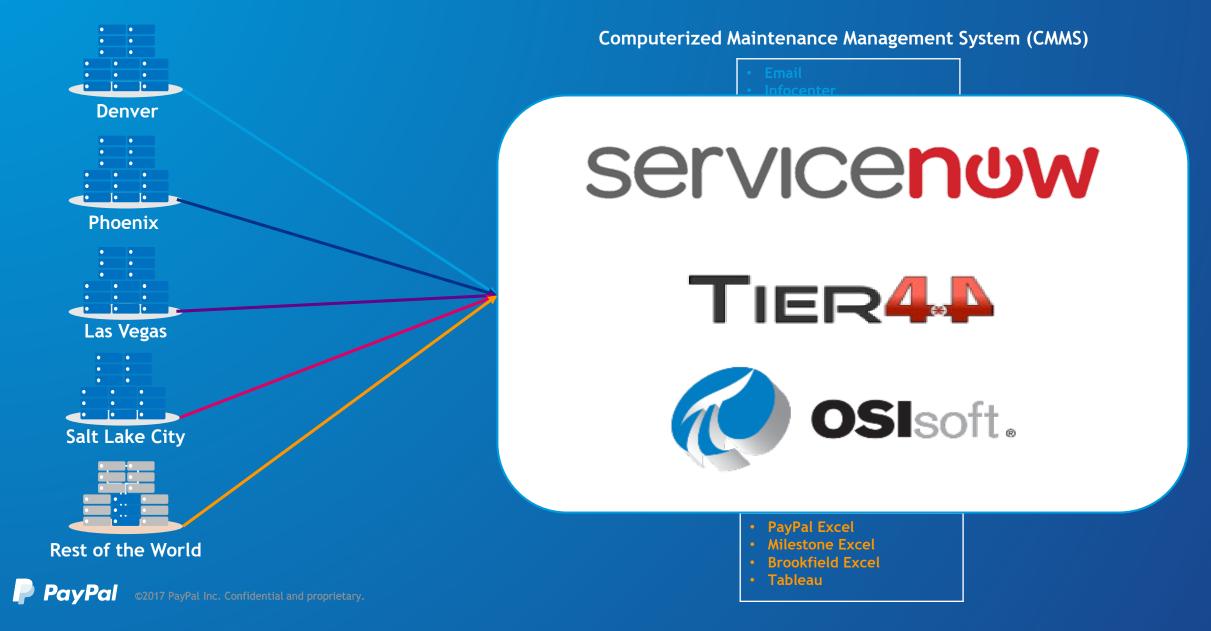
Phase 1 - Impact on data center efficiency The net result of the efficiency projects was an improvement of PUE from 1.85 to 1.49 (19%) across owned data centers.



Phase 2 - How we have traditionally managed our data centers Ticketing, monitoring, alarming, and data have historically been fragmented across multiple platforms



Phase 2 - How our data centers will be managed moving forward Ticketing, monitoring, alarming, and data will be consolidated into a single pane of glass



Phase 3 - Planned Future Improvements - "What's next?"

With the physical and application improvements in place we can begin to further automate our systems and predict future demand

Current Situation



Significant ongoing effort needed to continually audit assets to validate their location and position with racks

Solution

Power Strip Tagging Provides automated asset unit position within the rack saving time for audits and provides rack usage indicators.



RFID Tagging

Provides automated building level position and will be used to automatically receive assets and check them in/out of stock



Lead times for provisioning can be weeks if there is no space and power available at the time of a request Predictive analysis of space and power

Objective is to use data to more accurately project expected space and power usage



Projections would be used to automatically issue procurement or work tickets to stay ahead of the demand, reducing provisioning times from weeks to

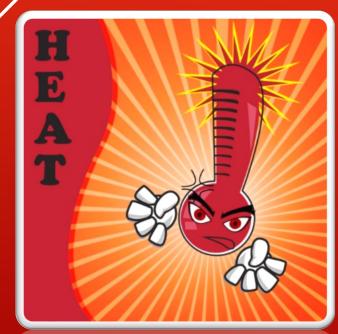


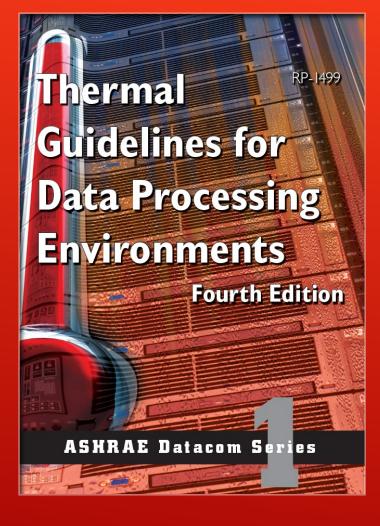


HOW HOT CAN I GO?

Intuit's Journey in Quincy Washington

Dave Breland Manager, Enterprise Data Center Facilities





FIRST- I NEEDED A GUIDE

ASHRAE Datacom Series formerly TC9.9

Thermal Guidelines for Data Processing Environments, Fourth Edition 13

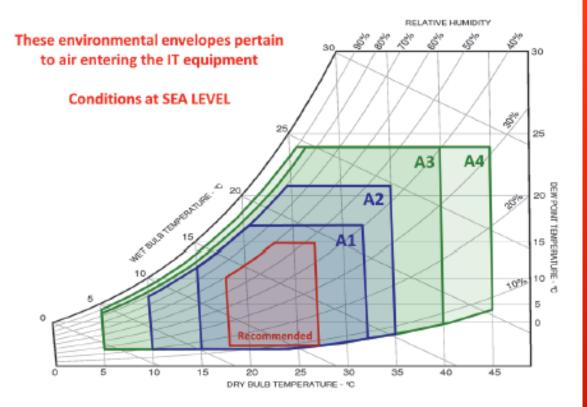


Figure 2.3 2015 recommended and allowable envelopes for ASHRAE Classes A1, A2, A3, and A4.

INSIDE ASHRAE'S DATACOM GUIDE

2.2.1 Environmental Class Definitions for Air-Cooled Equipment

Compliance with a particular environmental class requires full operation of the equipment over the entire allowable environmental range, based on nonfailure conditions.

Class A1: Typically a data center with tightly controlled environmental parameters (dew point, temperature, and RH) and mission-critical operations; types of products typically designed for this environment are enterprise servers and storage products.

| Table B.1 2015 Thermal Guidelines—I-P Version (SI Version in Table 2.1) | | | | | | | | |
|---|--|---|--|---|--|----------------------------------|---|--|
| | Equipment Environment Specifications for Air Cooling | | | | | | | |
| | | Product C | Operation ^{b,c} | | | Product Po | Product Power Off ^{c,d} | |
| Class ^a | Dry-Bulb Temperature ^{e,g} , °F | Humidity Range, Noncondensing ^{h,i,k,I} | Maximum Dew Point, ^k °F | Maximum Elevation ^{e, j, m} , ft | Maximum Rate , of Change ^f , °F/h | e Dry-Bulb Temperature, °F | Relative Humidity, ^k % | |
| Recon | nmended (Suitable | for all four classes; explor | re data center | metrics in this I | book for condition | ons outside this | range.) | |
| A1 to A4 | 64.4 to 80.6 | 15.8°F DP to 59°F DP and 60% rh | | | | | | |
| Allowa | ab le | | | | | | | |
| A1 | 59 to 89.6 | 10.4°F DP and 8% rh to 62.6°F DP and 80% rh | 62.6 | 10,000 | 9/36 | 41 to 113 | 8 to 80 | |
| A2 | 50 to 95 | 10.4°F DP and 8% rh to 69.8°F DP and 80% rh | 69.8 | 10,000 | 9/36 | 41 to 113 | 8 to 80 | |
| A3 | 41 to 104 | 10.4°F DP and 8% rh to 75.2°F DP and 85% rh | 75.2 | 10,000 | 9/36 | 41 to 113 | 8 to 80 | |
| A4 | 41 to 113 | 10.4°F DP and 8% rh to 75.2°F DP and 90% rh | 75.2 | 10,000 | 9/36 | 41 to 113 | 8 to 80 | |
| в | 41 to 95 | 8% to 82.4°F DP and 80% rh | 82.4 | 10,000 | N/A | 41 to 113 | 8 to 80 | |
| С | 41 to 104 | 8% to 82.4°F DP and 80% rh | 82.4 | 10,000 | N/A | 41 to 113 | 8 to 80 | |

2015 Thermal Guidelines—I-P Version (SI Version in Table 2.1)

Table B 1

* For potentially greater energy savings, refer to the section "Detailed Flowchart for the Use and Application of the ASHRAE Data Center Classes" in Appendix C for the process needed to account for multiple server metrics that impact overall TCO.

IN ENGLISH

Recommended vs. Allowable



HOW DO I KNOW?

Is there some way for me to know for sure?



Table 33. Environmental specifications

Fresh Air: temperature, humidity, altitude de-rating

within Dell Requirements

Continuous operation

32°C.

10°C to 35°C (50°F to 95°F) at 10% to 80% relative humidity with 26°C (78.8°F) maximum dew point (maximum wet bulb temperature). De-rate maximum allowable dry bulb temperature at 1°C per 300m above 950m (1°F per 547 ft above 3117 ft).

Dell states 10°C – 35°C while the ASHRAE Guideline allows 15°C – 32°C. ASHRAE Guidelines are well within Dell Requirements

PowerEdge R820

Technical Guide



Table 27. Environmental specifications

Fresh Air: temperature, humidity, altitude de-rating

Continuous operation 10°C to 35°C (50°F to 95°F) at 10% to 80% relative humidity with 26°C (78.8°F) maximum dew point (maximum wet bulb temperature). De-rate maximum allowable dry bulb temperature at 1°C per 300m above 950m (1°F per 547 ft above 3117 ft).

HP states 10°C – 35°C while the ASHRAE Guideline allows 15°C – 32°C. ASHRAE Guidelines are well within HP Requirements

HP ProLiant DL170e G6 Server - Overview

Product description



The DL170e G6 is optimized for efficiency, density and flexibility and can be serviced individually without impacting the operation of other server nodes sharing the same chassis. The DL2000 consists of up to 4 independent DL170e G6 servers in the 2U HP ProLiant e2000 G6 Chassis. The servers share power supplies and fans, providing greater power and cooling efficiencies.

Environmental specifications

| Item | Description |
|------------------------|---|
| Thermal output | 392 W/hr |
| (maximum operating) | |
| System inlet | 50° to 95° F (10° to 35° C) at sea level with an altitude derating of 1.8°F per every |
| temperature, operating | 1000 ft (1.0°C per every 305 m) above sea level to a maximum of 10,000 ft (3050 |
| | m), no direct sustained sunlight. Maximum rate of change is 18°F/hr (10°C/hr). The |
| | upper limit may be limited by the type and number of options installed. System |
| | performance may be reduced if operating with a fan fault or above 86°F (30°C). |

HP states 10°C – 35°C while the ASHRAE Guideline allows 15°C – 32°C. ASHRAE Guidelines are well within HP Requirements

System Inlet Temperature Operating

50° to 95° F (10° to 35° C) at sea level with an altitude derating of 1.8°F per every 1000 ft (1.0°C per every 305 m) above sea level to a maximum of 10,000 ft (3050 m), no direct sustained sunlight. Maximum rate of change is 18°F/hr (10°C/hr). The upper limit may be limited by the type and number of options installed.

System performance may be reduced if operating with a fan fault or above 86°F (30°C).

HP ProLiant DL2000 server technologies



Cisco states 0°C – 40°C while the ASHRAE Guideline allows 15°C – 32°C. ASHRAE Guidelines are well within Cisco Requirements

Cisco Nexus 9396PX



Environmental Specifications

| Environment | | Specification |
|-------------------|-------------------------------|--------------------------------------|
| Temperature | Ambient operating temperature | 32 to 104°F (0 to 40°C) |
| | Ambient nonoperating | -40 to 158° F (-40 to 70° C) |
| Relative humidity | Ambient (noncondensing) | 5 to 95% |
| Altitude | Operating | 0 to 13,123 feet (0 to 4,000 meters) |

Cisco Nexus 7000 18-Slot Switch

Compare Models View 3D Model

Cisco states 0°C – 40°C while the ASHRAE Guideline allows 15°C – 32°C. ASHRAE Guidelines are well within Cisco Requirements



Climatic Environment

Table 10 summarizes the climatic environment for the Cisco Nexus 7000 4-, 9-, 10-, and 18-Slot chassis.

Table 10. Climatic Environment

| Item | Description | | | |
|-------------|---|---------------------------------|---|-------------------------------------|
| | Cisco Nexus 7000 4-Slot Chassis | Cisco Nexus 7000 9-Slot Chassis | Cisco Nexus 7000 10-Slot Chassis | Cisco Nexus 7000 18-Slot Chassis |
| Temperature | erature erature erature e Operating: 32 to 104°F (0 to 40°C) eshort term: 23 to 131°F (-5 to 55°C)* eshort term: 23 to 131°F (-40 to 70°C)* Nonoperating: -40 to 158°F (-40 to 70°C)* Note: e Chassis external thermal requirements are defined in the GR-63-CORE Network Equipment Building Standards (NEBS) specification published by Telcordia Technologies in Section 4.1.2, Operating Temperature and Humidity Criteria. Short term refers to a period of not more than 96 consecutive hours and a total of not more than 15 days in 1 year (a total of 360 hours in any given year, but no more than 15 occurrences during that 1-year period). * 131°F (55°C) and 25% relative humidity for equipment shelves that fill less than half the frame (length) | | , | |
| Humidity | Relative humidity (nonoperating): 5 to 95% Relative humidity (operating): 5 to 90%, no Note: An ambient relative humidity between 4 interference from static discharges. | ncondensing | roblems, to provide an operating time buffer in t | ne event of failures, and to reduce |

NetApp Disk Shelves and Storage Media Technical Specifications

NetApp states 0°C – 40°C while the ASHRAE Guideline allows 15°C – 32°C. ASHRAE Guidelines are well within

NetApp Requirements



Environmental Specifications

| Environment | | Specification |
|-------------------|-------------------------------|--------------------------------------|
| Temperature | Ambient operating temperature | 32 to 104°F (0 to 40°C) |
| | Ambient nonoperating | -40 to 158° F (-40 to 70° C) |
| Relative humidity | Ambient (noncondensing) | 5 to 95% |
| Altitude | Operating | 0 to 13,123 feet (0 to 4,000 meters) |

Apple states 0°C – 35°C while the ASHRAE Guideline allows 15°C – 32°C. ASHRAE Guidelines are well within Apple Requirements

Electrical and operating requirements

- Line voltage: 100-240V AC
- Frequency: 50Hz to 60Hz, single phase
- Maximum continuous power: 450W
- Operating temperature: 50° to 95° F (10° to 35° C)
- Relative humidity: 5% to 95% noncondensing
- Maximum altitude: 16,400 feet (5000 meters)
- Typical acoustical performance, sound pressure level (operator position): 12 dBA at idle





Pure Storage states 5°C – 35°C while the ASHRAE Guideline allows 15°C – 32°C. ASHRAE Guidelines are well within Pure Storage Requirements

ENVIRONMENTAL

| //M10, //M20, //M50, //M70 | RANGE |
|----------------------------|-------------|
| Operating Temperature | 5° to 35° C |
| Non-operating Temperature | 0° to 60° C |
| Operating Humidity | 10 – 80% |
| Non-operating Humidity | 5 – 95% |
| | |

| NOTES |
|---|
| Derate 1 C per 300 m above 950 m |
| |
| Non-condensing |
| At a maximum temperature of 33° C. Non-condensing, web bulb 33° C. |

PURESTORAGE*



Although OSHA does not have a particular regulation or standard that covers hightemperature environments, the General Duty Clause, Section 5(a)(1) of the Occupational Safety and Health Act of 1970 (OSHA 2012b), requires each employer to "furnish to each of his employees employment and a place of employment which are free from recognized hazards that are causing or are likely to cause death or serious physical harm." OSHA has interpreted this rule such that employers shall provide means and methods that will reduce the likelihood of worker heat stress. These means or methods may include issuing personal protective equipment (PPE), minimizing exposure through frequent breaks, frequent hydration, and developing a heat stress program. There are various manufacturers that produce PPE for hot working environments.

Table E.1 Permissible Heat Exposure Threshold Limit Value (TLV) (ACGIH 1992)

| Work/Rest Regimen - | | Work Load ¹ | |
|----------------------------------|---------------|------------------------|---------------|
| | Light | Moderate | Heavy |
| Continuous work | 30.0°C (86°F) | 26.7°C (80°F) | 25.0°C (77°F) |
| 75% work, 25% rest, each hour | 30.6°C (87°F) | 28.0°C (82°F) | 25.9°C (78°F) |
| 50% work, 50% rest, each hour | 31.4°C (89°F) | 29.4°C (85°F) | 27.9°C (82°F) |
| 25% work, 75% rest, each hour | 32.2°C (90°F) | 31.1°C (88°F) | 30.0°C (86°F) |

1. Values are in °C and °F (wet-bulb globe temperature [WBGT]).

Appendix E OSHA and Personnel Working in High Air Temperatures



INTUIT'S JOURNEY IN QUINCY WASHINGTON

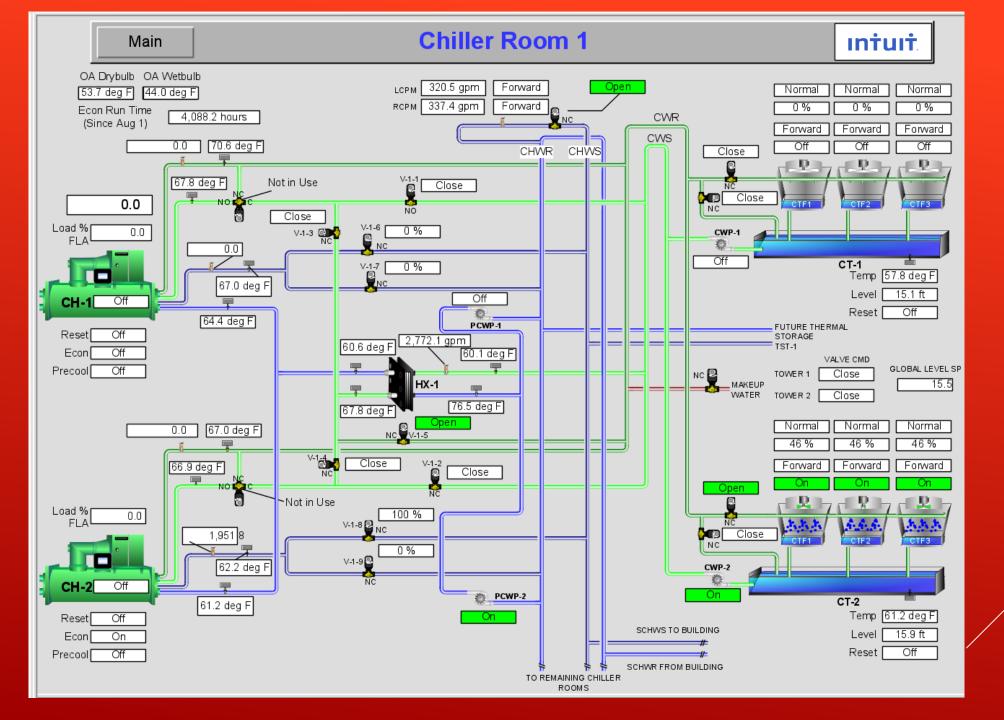
THEN

- Original Design 72°f top of Rack
- CWS Temp 47°f
- Economizer Start OAWBT 45°f with a CWST of <47.5°f.
- Economizer Stop 48.5°f CWST
- Averaged 48% of year on economizer at current load

NOW

- Current Control 78.5°f top of Rack Warn 80°f Alarm
- CWS Temp 61°f
- Economizer Start OAWBT 58°f with a CWST of <61.5°f.
- Economizer Stop 64°f CWST
- Now average 65% on water side economizer

| Main Econ (Sir | Run Time 4,088.2 hours Chilled Water Plant Setup-Lead OA Drybulb OA Wetbulb (44.2 deg F) (44.2 deg F) |
|--|--|
| SELECT ONE COM CHW BYP ASS WEST (A) CI WEST (B) CI DESIRED OPERATING RANK RANK | n 363.0 gpm Forward 5.5 LEVEL MAKEUP SP STAGE UP A CHILLER, IF BYP ASS FLOW < 1,000.0 gpm For 300.0 secs |
| CHILLER 1 3 3.0 | Off #1 False False Normal Off STARTECON AT 0A-WBT < 58.0 deg F AND HX CHWLWT < 61.5 FOR 300.0 SECS |
| | Off Off Off Off Off Off For 5 minutes Off #2 False False Normal On Off Stop e con hx chwLwt > 64.0 For 5 minutes Off Off Off Off Off Off For 5 minutes Off Off Off Off Off For 5 minutes |



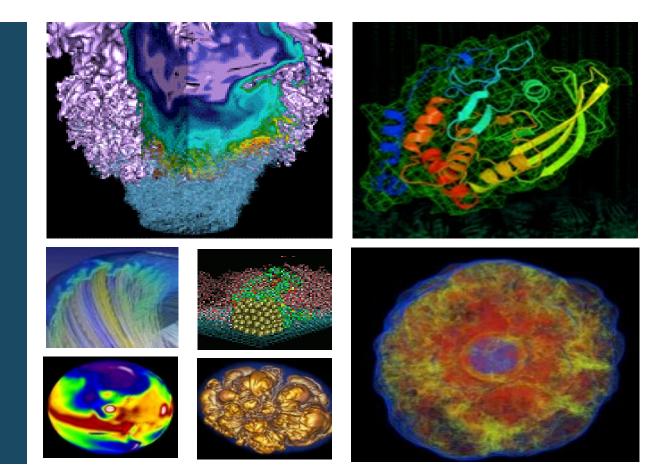
One of Our Cooling Plants



- ► Balance data halls using Purkay instruments and Tileflow Software, to ensure cold aisle temps ≤ 78.5°f. at top of rack.
- ► Raised CWST to get all of the CRAH control valves at their mid range.
- Operators monitor weather patterns and redundant cooling tower storage to extend economizer hours when possible.
- My biggest bang is staying on my economizer as long as possible.
- With my water side economizer, it's as far as I can go. Most later version data centers around me are using 100% outside air or probably half have gone to the Munters Evaporative Cooling units.

SO THIS IS HOW WE HAVE DETERMINED HOW HOT WE CAN GO

National Energy Research Scientific Computing Center (NERSC) Shyh Wang Hall





7x24 – Shaping The Future Phoenix, Arizona October 17, 2017 Dale Sartor, PE

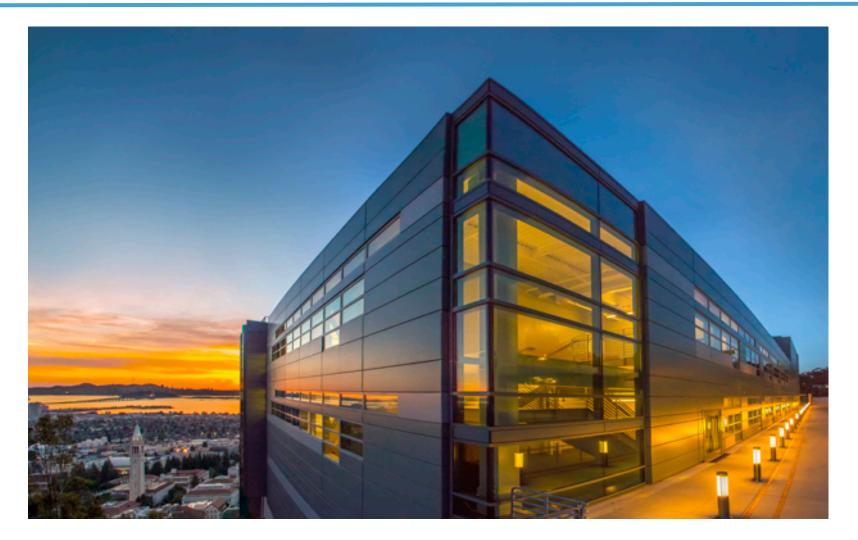




Version: 090817

Shyh Wang Hall, the New Home for NERSC











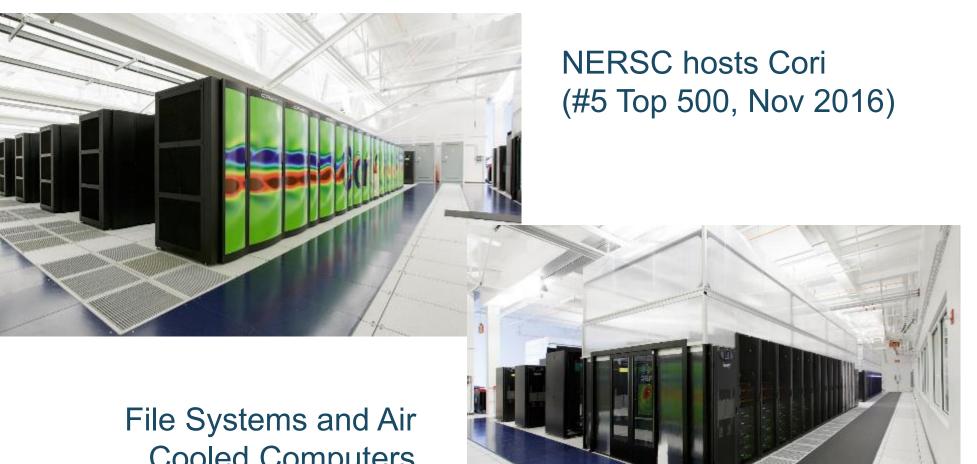
- 142,000 square feet total
- 7 MW IT load to start, then up to 17, then ???
- IT load will dominate building
- 4 large AHUs for air-cooled loads
- 4 cooling towers for water cooled loads
- Water-cooled supercomputers
- Air and water side economizers
- IT loads cooled without compressors
- Air-side heat recovery for heating offices





High and Low Density Computing







Cooled Computers

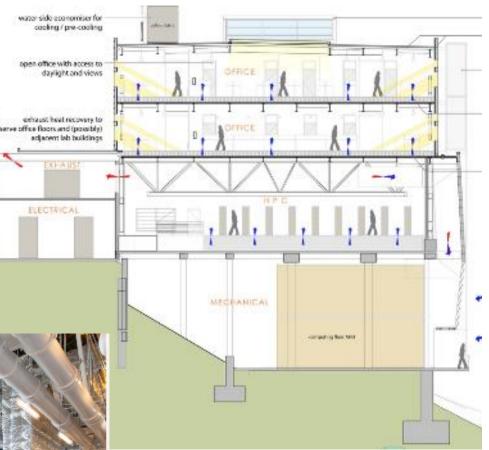


Free Cooling - Air System Design



- 2 MW Air Cooling (17 MW max)
- Annual PUE less than 1.1
- Air-side economizer
- Direct evaporative cooling for humidification & cooling





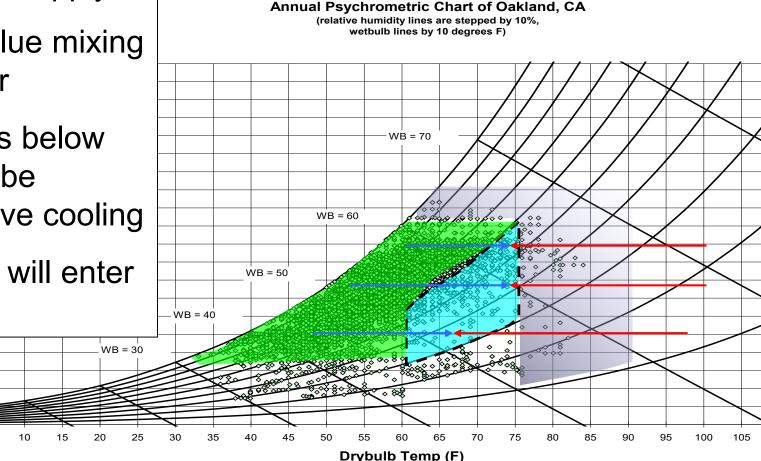




Outside Air Based Design



- 1. Blue = original design supply
- 2. Green can become blue mixing return and outdoor air
- Most of the conditions below and right of blue can be satisfied w/ evaporative cooling
- 4. Hot and humid hours will enter the "allowable" range







Free Cooling - Water Based Design

- 10 MW liquid cooling (20 MW max)
- Annual PUE less than 1.1
- Closed-loop treated cooling water from cooling towers
- Headers, valves and caps for modularity and flexibility











Real Life Results

- Performance
- Opportunities for improvement/optimization
- Lessons learned

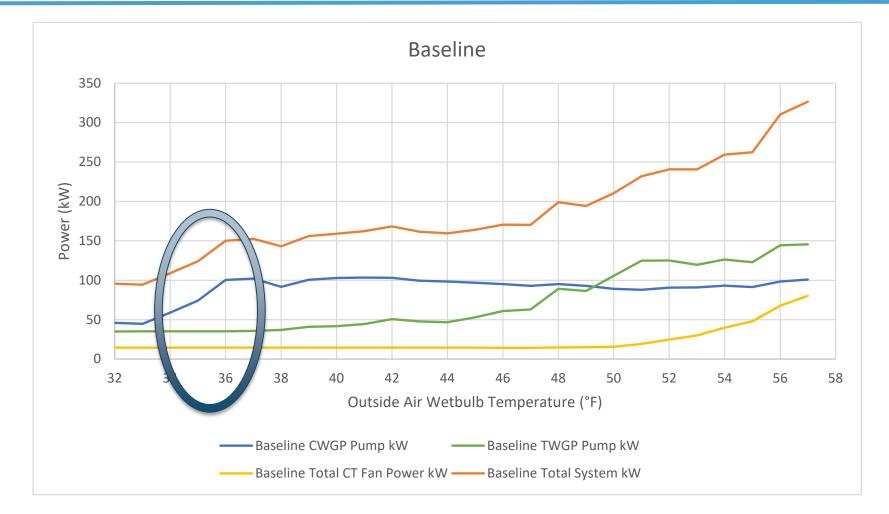






Liquid Cooling Performance Baseline





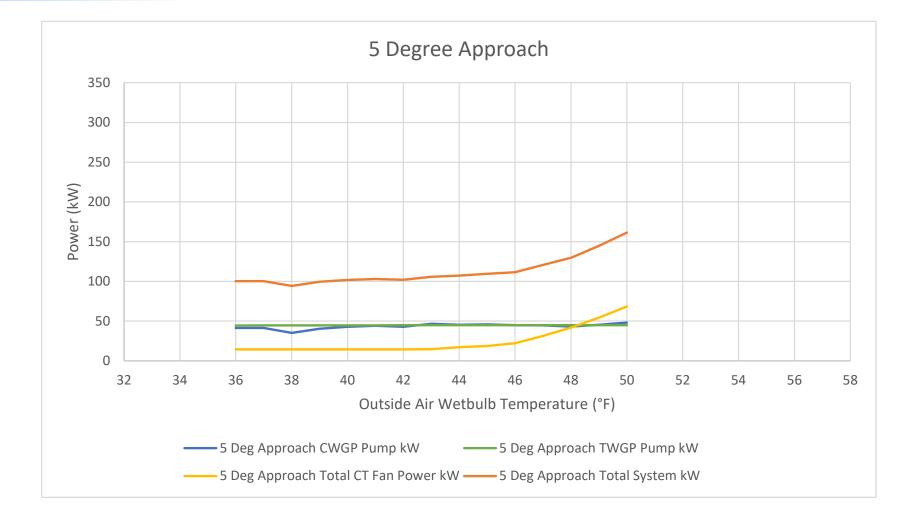






Liquid Cooling Performance Balanced













FRKELEY I

Taking Full Advantage of Recommended Ranges

- Allowable ranges have expanded
- Trade off between high humidity and high temperature
 - > We chose high humidity
- Allowing dryer air would save water

Linric Company Psychrometric Chart, www.linric.com



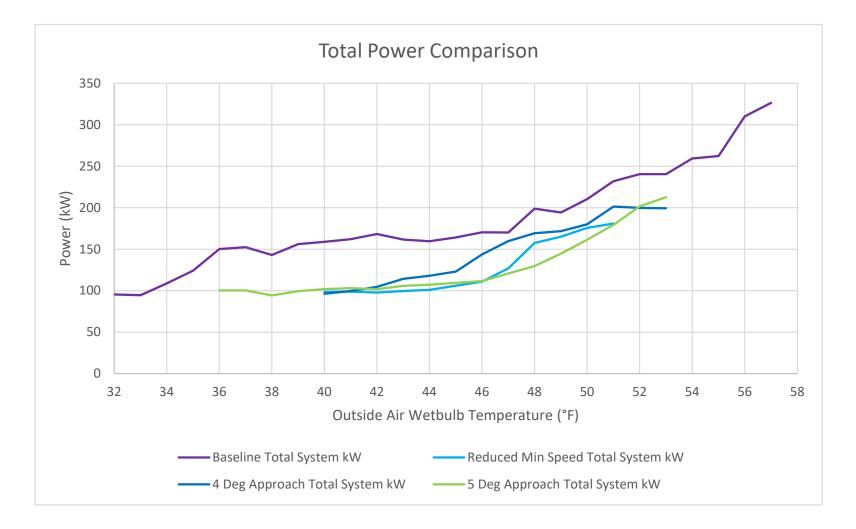




DRY BULB TEMPERATURE - °F

Cooling System Optimization





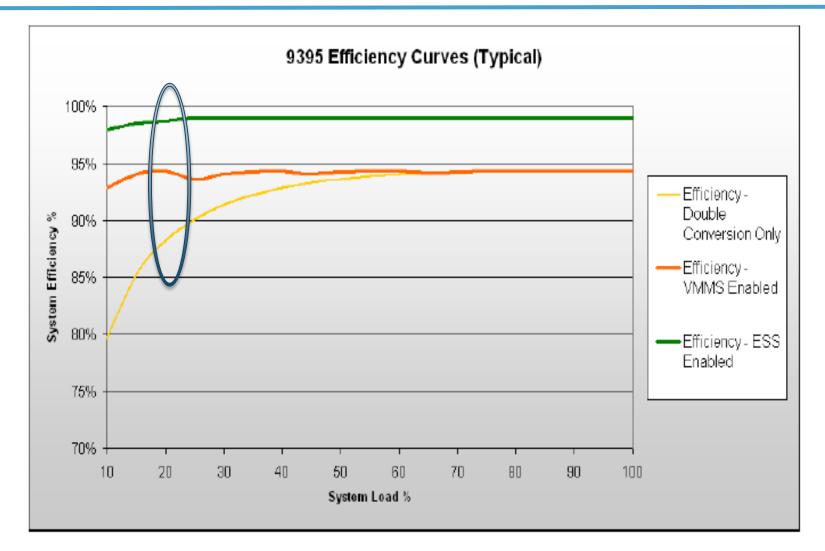






UPS Double Conversion Losses













| | | Energy Savings (kWh) | | Water Savings | Cost Savings | PUE |
|-----------------|--|----------------------|----------|---------------|--------------|-----------|
| | Measure Title | Estimated | Verified | Gallons | \$ | Reduction |
| <u>Controls</u> | | | | | | |
| 1 | Optimize Cooling Tower Fan and Pump Controls | - | 360,000 | 100,000 | \$ 20,880 | 0.007 |
| 2 | Optimize Closed Loop Pump Control | 240,000 | - | 110,000 | \$ 13,920 | 0.005 |
| 3 | Optimize AHU SAT and Flow Control | 300,000 | - | _ | \$ 17,400 | 0.006 |
| 4 | Reset Cooling Water Supply Temperature | 600,000 | - | 220,000 | \$ 34,800 | - |
| 5 | Install Firmware to Enable ESS Mode for UPSs | 190,000 | - | 65,000 | \$ 11,020 | 0.004 |
| Phys | Physical Projects | | | | \$ - | |
| 6 | Cold Aisle Partial Containment | 100,000 | - | - | \$ 5,800 | 0.002 |

| Total | 1,400,000 | 400,000 | 500,000 | \$ | 100,000 | 0.025 |
|-------|-----------|---------|---------|----|---------|-------|
|-------|-----------|---------|---------|----|---------|-------|

| | IT kWh | 48,200,000 | Extrapolated based on typical operation |
|-----------|---|------------|---|
| 14/ | Total Non-IT kWh | 3,200,000 | Does not include CRAY fans |
| κW | PUE | 1.07 | |
| | Estimated Post-Case PUE | 1.04 | |
| GINEERING | Savings as a Fraction of Cooling System kWh | 56% | |





K



- 1. Identify opportunities
- 2. Identify critical control boundaries
- 3. Try it out!
- 4. Verify using trend data
- 5. Repeat until optimized

All parties work closely throughout the project









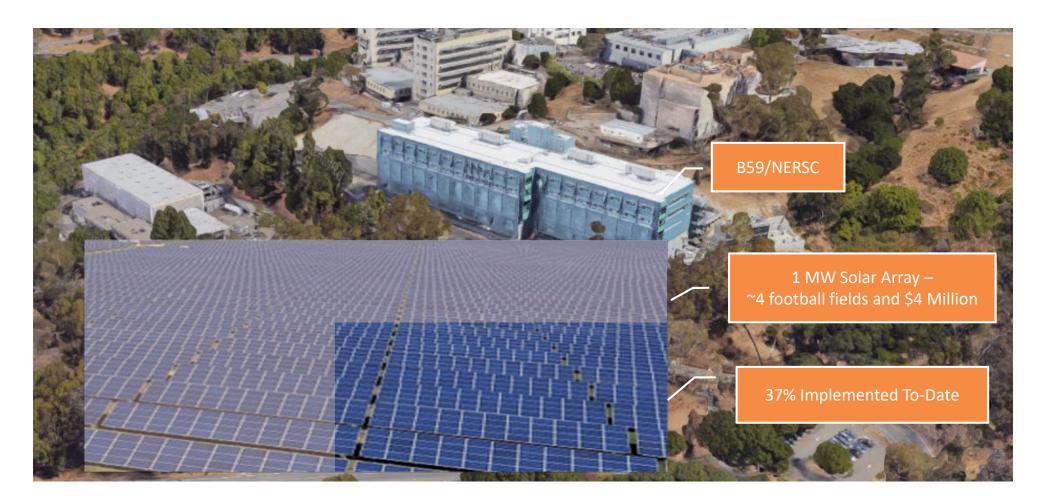
- Involve full team from measure development through implementation
- Question everything
- Incremental, iterative process can be most effective
- Don't underestimate interconnectivity of systems
- A sub 1.1 PUE data center can still be improved!





Our Identified Savings are Equivalent to a 1 MW Solar Array













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http://datacenters.lbl.gov/





